

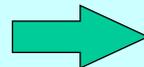
## Chapter 12

# Gravity currents, bores and flow over obstacles



## Hydraulic theory

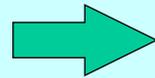
- Now we examine some simple techniques from the theory of hydraulics to study a range of small scale atmospheric flows, including **gravity currents, bores (hydraulic jumps) and flow over orography.**





## Bores

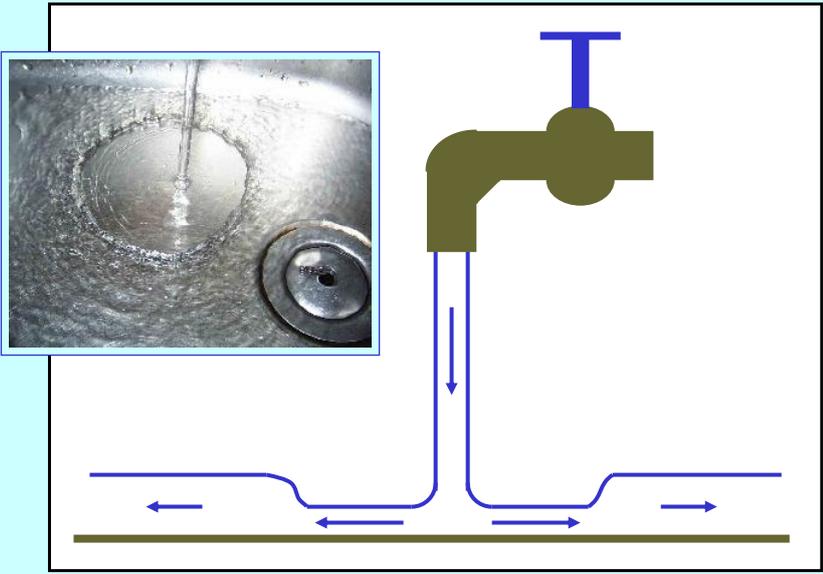
- The more familiar examples of bores occur on water surfaces.
- Examples are:
  - bores on tidal rivers,
  - quasi-stationary bores produced downstream of a weir, and
  - the bore produced in a wash basin when the tap is turned on and a laminar stream of water impinges on the bottom of the basin.
- Perhaps the best known of atmospheric bores is the so-called '**morning glory**' of northern Australia.



## Bores on rivers



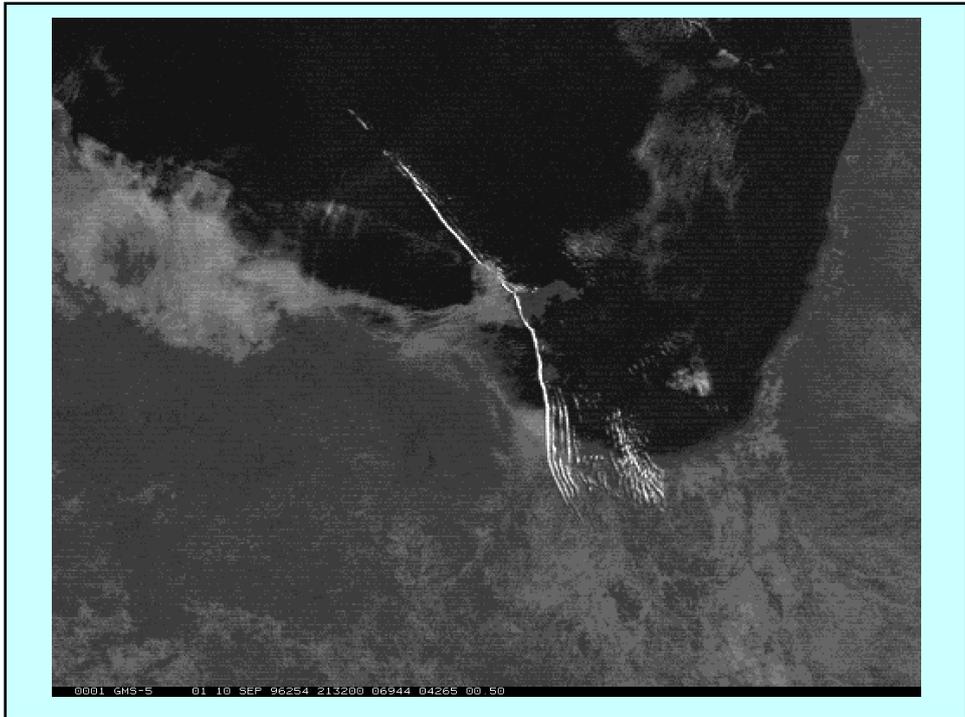
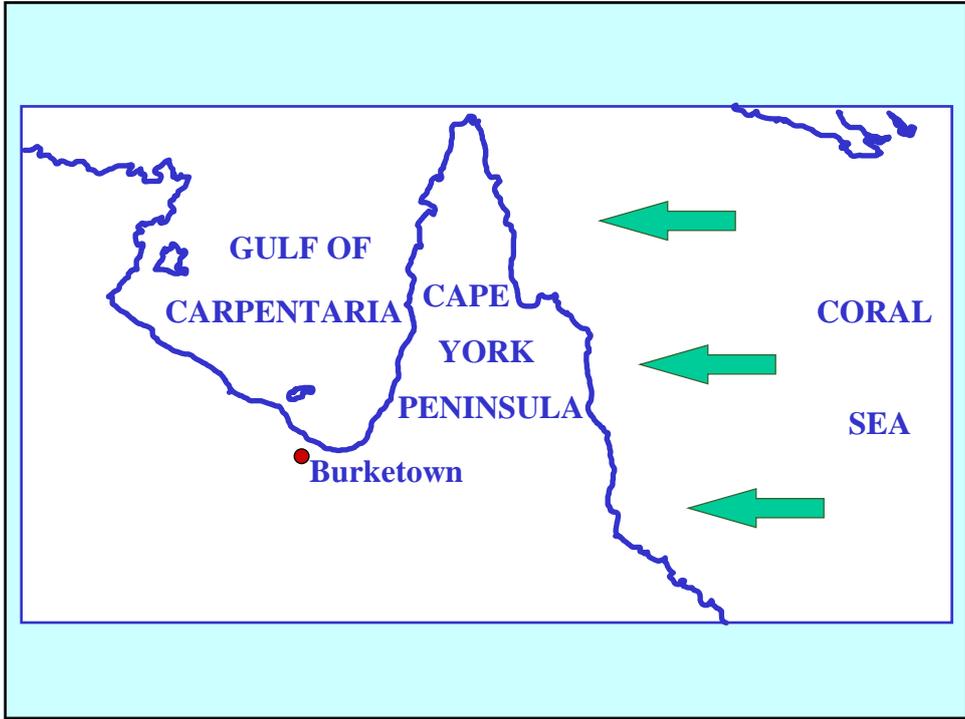
**Mascaret bore, France**



- The morning glory is produced by the **collision of two sea breezes** over Cape York Peninsula and is formed on a low-level stable layer, typically 500 m deep.
- The bore is regularly accompanied by **spectacular roll clouds**.
- Similar phenomena occur elsewhere, but not with such regularity in any one place.
- Another atmospheric example is when a **stratified airstream flows over a mountain ridge**.
- Under certain conditions a phenomenon akin to a bore, or hydraulic jump, may occur in the lee of the ridge.



The “Morning Glory”



## Bernoulli's theorem

- Euler's equation for an inviscid rotating flow on an f-plane is

$$\partial_t \mathbf{u} + \mathbf{u} \cdot \nabla \mathbf{u} + f \mathbf{k} \wedge \mathbf{u} = -(1/\rho) \nabla p_T - \mathbf{g}$$

Use the vector identity

$$\mathbf{u} \cdot \nabla \mathbf{u} = \nabla \left( \frac{1}{2} \mathbf{u}^2 \right) + \boldsymbol{\omega} \wedge \mathbf{u}$$

$$\boldsymbol{\omega} = \text{curl } \mathbf{u}$$

➔ 
$$\partial_t \mathbf{u} + \nabla \left( \frac{1}{2} \mathbf{u}^2 + gz \right) + (1/\rho) \nabla p_T + (f \mathbf{k} + \boldsymbol{\omega}) \wedge \mathbf{u} = 0$$

Assume steady flow ( $\partial_t \mathbf{u} = \mathbf{0}$ ) and a homogeneous fluid ( $\rho = \text{constant}$ ): then

$$\mathbf{u} \cdot \nabla \left( \frac{1}{2} \mathbf{u}^2 + p_T / \rho + gz \right) = 0$$

$$\mathbf{u} \cdot \nabla \left( \frac{1}{2} \mathbf{u}^2 + p_T / \rho + gz \right) = 0$$

For the **steady flow of homogeneous, inviscid fluid**, the quantity  $H$  given by

$$H = \frac{1}{2} \mathbf{u}^2 + p_T / \rho + gz$$

is a constant along a streamline.

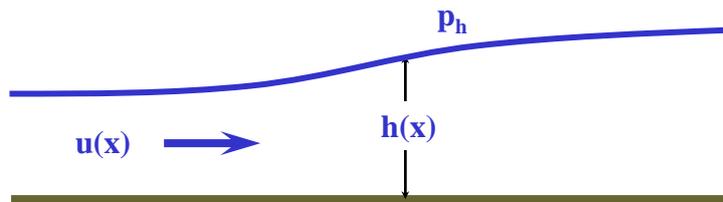
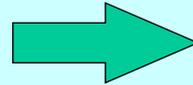
- This is **Bernoulli's theorem**.
- The quantity  $H$  is called the **total head** along the streamline and is a measure of the total energy per unit volume on that streamline.
- Note: it may be that a flow in which we are interested is unsteady, but can be made steady by a **Galilean coordinate transformation**.
- Then Bernoulli's theorem can be applied in the transformed frame.

## Flow force

- Consider steady motion of an inviscid rotating fluid in two dimensions.
- In flux form the x-momentum equation is

$$\partial_x(\rho u^2) + \partial_z(\rho u w) - \rho f v = -\partial_x p_T$$

Consider the motion of a layer of fluid of variable depth  $h(x)$ ; see next figure: =>



**Integrating**  $\partial_x(\rho u^2) + \partial_z(\rho u w) - \rho f v = -\partial_x p_T$  with respect to  $z$

$$\int_0^h \partial_x [\rho u^2 + p_T] dz = -[\rho u w]_0^h + \int_0^h f \rho v dz$$

$$\begin{aligned} \frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz - [\rho u^2 + p_T]_h \frac{\partial h}{\partial x} \\ = -(\rho u^2)_h \frac{\partial h}{\partial x} + f \int_0^h \rho v dz. \end{aligned}$$

$$\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz - [\rho u^2 + p_T]_h \frac{\partial h}{\partial x} = -(\rho u^2)_h \frac{\partial h}{\partial x} + f \int_0^h \rho v dz.$$

In particular, if  $f = 0$ , then

$$\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_h \frac{\partial h}{\partial x}$$

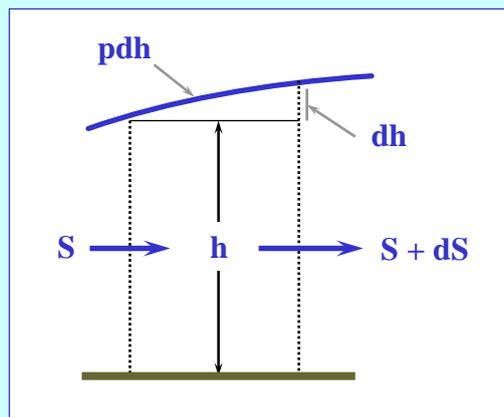
Define

$$S = \int_0^h [\rho u^2 + p_T] dz$$

then

$$\frac{\partial S}{\partial x} = p_h \frac{\partial h}{\partial x}$$

Considering the control volume =>



$$\frac{\partial S}{\partial x} = p_h \frac{\partial h}{\partial x} \quad \rightarrow \quad dS = p_h dh$$

$$\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_h \frac{\partial h}{\partial x} \quad \text{and} \quad S = \int_0^h [\rho u^2 + p_T] dz$$

**Include a frictional force,  $-\rho D$ , per unit volume**

**then** 
$$\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_h \frac{\partial h}{\partial x} - \int_0^h \rho D dz$$



$$dS = p_h dh - D^* dx$$

**call  $D^*$**

**Two useful deductions of  $\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_h \frac{\partial h}{\partial x}$  are:**

1. **If  $h = \text{constant}$ ,**  $\int_0^h [\rho u^2 + p_T] dz = \text{const tan t}$
2. **If  $p_h = \text{constant}$ ,**  $\int_0^h [\rho u^2 + p_T] dz - p_h h = \text{const tan t}$

## Summary of important results

### Bernoulli's theorem

**For the steady flow of homogeneous, inviscid fluid**

$$H = \frac{1}{2} \mathbf{u}^2 + p_T / \rho + gz$$

**is a constant along a streamline.**

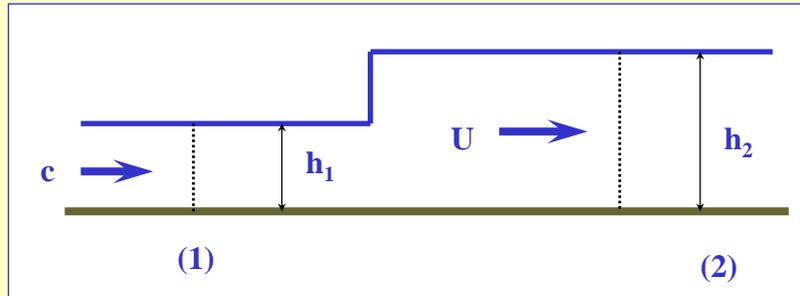
### Flow force

$$S = \int_0^h [\rho u^2 + p_T] dz$$

1. **If  $h = \text{constant}$ ,**  $\int_0^h [\rho u^2 + p_T] dz = \text{const tan t}$
2. **If  $p_h = \text{constant}$ ,**  $\int_0^h [\rho u^2 + p_T] dz - p_h h = \text{const tan t}$

## Theory of hydraulic jumps, or bores

- We idealize a jump by an abrupt transition in fluid depth.



Schematic diagram of a hydraulic jump, or bore

- Express mathematically in terms of the **Heaviside step function**

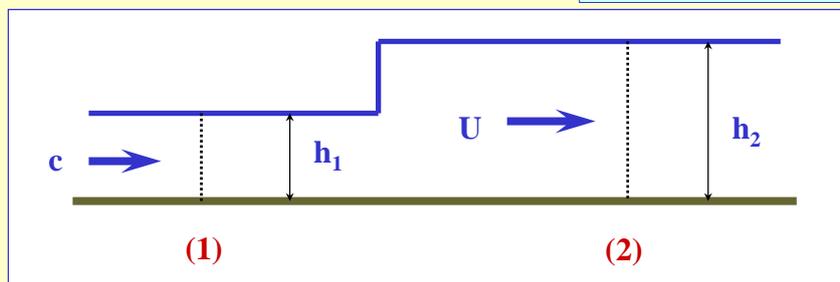
$$h(x) = h_1 + (h_2 - h_1)H(x)$$

$$\frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_h \frac{\partial h}{\partial x}$$

$$\Rightarrow \frac{\partial}{\partial x} \int_0^h [\rho u^2 + p_T] dz = p_a (h_2 - h_1) \delta(x)$$

atmospheric pressure

Dirac delta function



**Integrate with respect to x between (1) and (2) =>**

$$\int_0^{h_2} [\rho u^2 + p_T]_2 dz = \int_0^{h_1} [\rho u^2 + p_T]_1 dz + p_a (h_2 - h_1)$$

$$\int_0^{h_2} [\rho u^2 + p_T]_2 dz = \int_0^{h_1} [\rho u^2 + p_T]_1 dz + p_a (h_2 - h_1)$$

$$\Rightarrow U^2 h_2 + \frac{1}{2} g h_2^2 = c^2 h_1 + \frac{1}{2} g h_1^2$$

We have used the fact that the flow at positions (1) and (2) is horizontal and therefore the pressure is hydrostatic.

Continuity of mass (and hence volume) gives

$$c h_1 = U h_2$$

Solve for  $c^2$  and  $U^2$  in terms of  $h_1$  and  $h_2$

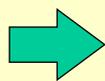
$$\Rightarrow c = \left[ \frac{1}{2} g (h_1 + h_2) h_2 / h_1 \right]^{1/2}$$

and  $U = \left[ \frac{1}{2} g (h_1 + h_2) h_1 / h_2 \right]^{1/2}$

Use Bernoulli's theorem =>

The change in total head along the surface streamline is

$$\begin{aligned} \delta H &= p_a + \frac{1}{2} \rho U^2 + \rho g h_2 - p_a - \frac{1}{2} \rho c^2 - \rho g h_1, \\ &= \frac{1}{2} \rho \left[ \frac{1}{2} g (h_1 + h_2) \left[ \frac{h_1}{h_2} - \frac{h_2}{h_1} \right] + 2g (h_2 - h_1) \right], \\ &= \frac{1}{4} \rho g \left[ \frac{(h_1^2 - h_2^2)(h_2 + h_1)}{h_1 h_2} + 4(h_2 - h_1) \right], \\ &= \frac{1}{4} \rho g (h_1 - h_2)^3 \cdot \frac{1}{h_1 h_2} \\ &< 0 \text{ if } h_1 < h_2 . \end{aligned}$$



Energy is lost at the jump

## Energy loss

- The energy lost supplies the source for the turbulent motion at the jump that occurs in many cases.
- For weaker bores, the jump may be accomplished by a series of smooth waves.
- Such bores are termed **undular**.
- In these cases the energy loss is radiated away by the waves. See Lighthill, 1978, §2.12.

- It follows from

$$\delta H = \frac{1}{4} \rho g (h_1 - h_2)^3 \cdot \frac{1}{h_1 h_2} < 0 \text{ if } h_1 < h_2 .$$

that the depth of fluid must **increase**, since a decrease would require an energy supply.

Then  $c = [\frac{1}{2} g (h_1 + h_2) h_2 / h_1]^{1/2}$  and

$$U = [\frac{1}{2} g (h_1 + h_2) h_1 / h_2]^{1/2}$$

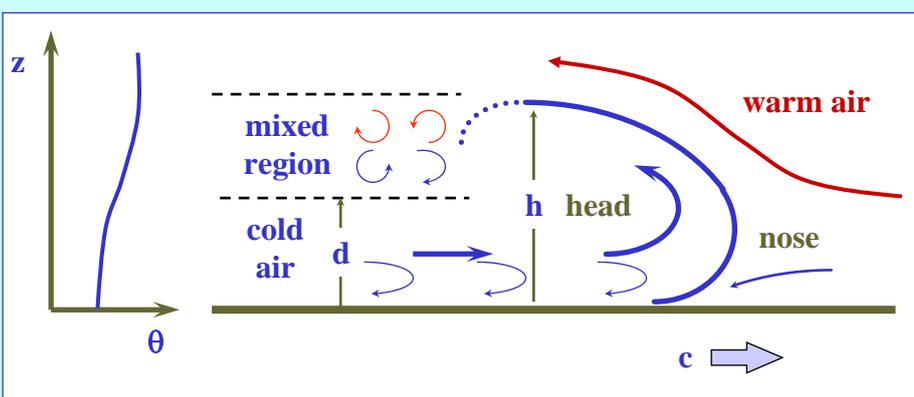


$$c > \sqrt{gh_1} \quad \text{and} \quad U < \sqrt{gh_2}$$

$$c > \sqrt{gh_1} \quad \text{and} \quad U < \sqrt{gh_2}$$

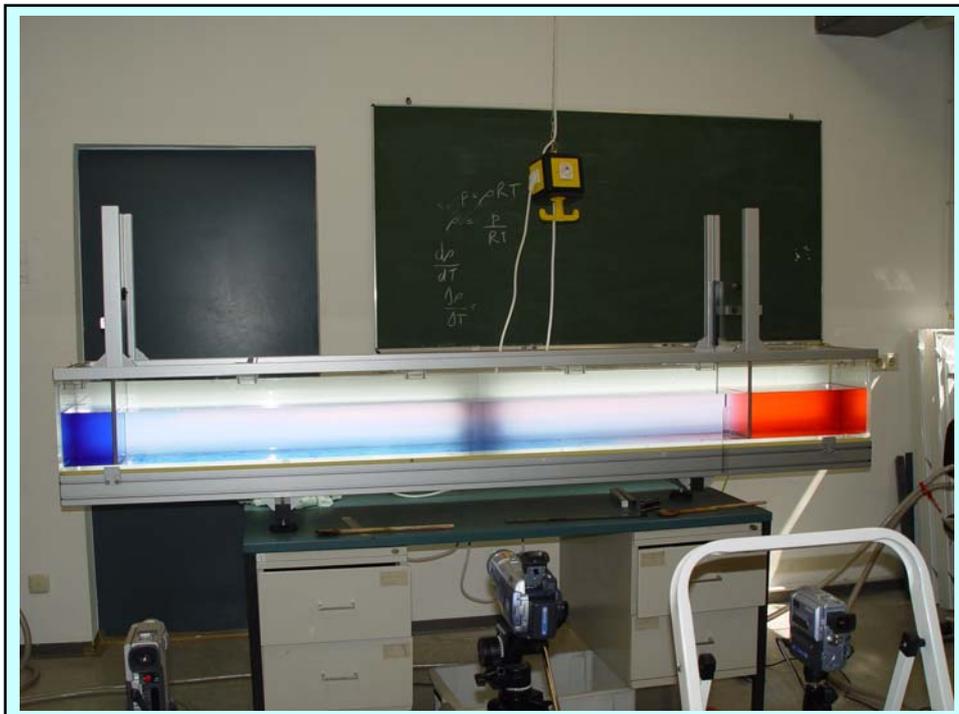
- Recall that  $\sqrt{gh}$  is the phase speed of long gravity waves on a layer of fluid of depth  $h$ .
- On the upstream side of the bore, gravity waves cannot propagate against the stream whereas, on the downstream side they can.
- Accordingly we refer to the flow upstream as **supercritical** and that downstream as **subcritical**.
- These terms are analogous to **supersonic** and **subsonic** in the theory of gas dynamics.

### Theory of gravity currents



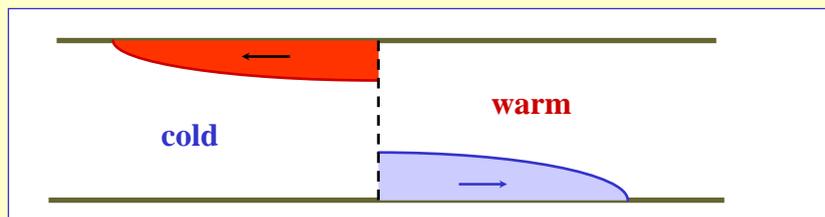
Schematic diagram of a steady gravity current

# Haboob



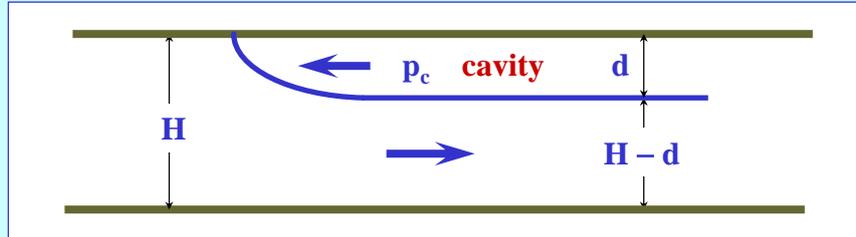
Show movies

- There is a certain symmetry between a gravity current of dense fluid that moves along the lower boundary in a lighter fluid, and a gravity current of light fluid that moves along the upper boundary of a denser fluid.



- The latter type occurs, for example, in a cold room when the door to a warmer room is opened.
- Then, a **warm gravity current runs along the ceiling of the cold room** and a **cold gravity current runs along the floor of the warm room**.

## Cavity flow



- The simplest flow configuration of these types is the flow of an **air cavity** into a long closed channel of fluid.
- In this case we can neglect the motion of the air in the cavity to a good first approximation.
- In practice the cavity will move steadily along the tube with speed  $c$ , say.

## Summary of important results

### Bernoulli's theorem

For the **steady flow of a homogeneous, inviscid fluid**

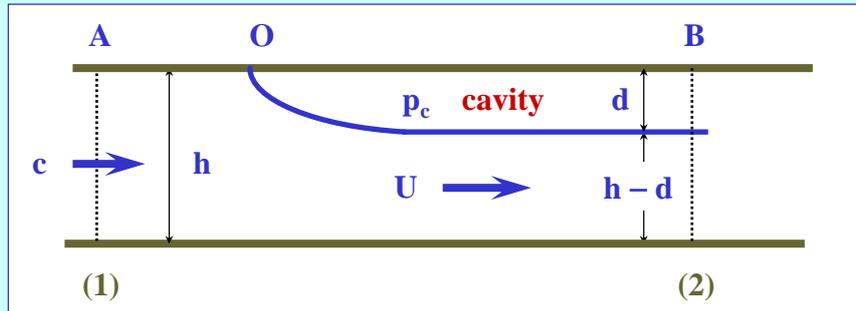
$$H = \frac{1}{2} \mathbf{u}^2 + p_T / \rho + gz$$

is a constant along a streamline.

### Flow force

$$S = \int_0^h [\rho u^2 + p_T] dz$$

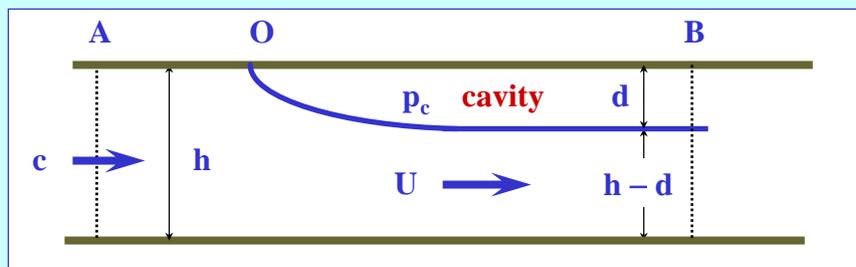
1. If  $h = \text{constant}$ ,  $\int_0^h [\rho u^2 + p_T] dz = \text{const}$   $\forall t$
2. If  $p_h = \text{constant}$ ,  $\int_0^h [\rho u^2 + p_T] dz - p_h h = \text{const}$   $\forall t$



- Choose a frame of reference in which the **cavity is stationary**  
=> the fluid upstream of the cavity moves towards the cavity with speed  $c$ .
- Apply Bernoulli's theorem along the streamline from **A to O**,  
=> since  $z = h = \text{constant}$ ,

$$p_A + \frac{1}{2} \rho c^2 = p_c$$

Note: O is a stagnation point => the pressure there is equal to the cavity pressure.

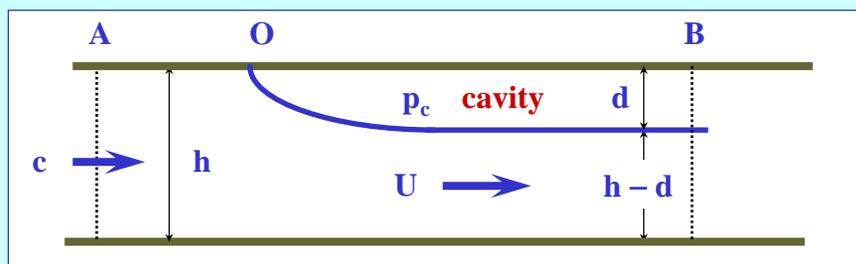


- Along the section between **A and O**

$$\int_0^h [\rho u^2 + p_T]_A dz = \int_0^h [\rho u^2 + p_T]_O dz$$

- Along the section between **O and B**

$$\int_0^h [\rho u^2 + p_T]_O dz - p_c h = \int_0^{h-d} [\rho u^2 + p_T]_B dz - p_c (h - d)$$



$$\int_0^h [\rho u^2 + p_T]_A dz = \int_0^{h-d} [\rho u^2 + p_T]_B dz + p_c d$$

At A and B where the flow is parallel (i.e.  $w = 0$ ), the pressure is hydrostatic

$$\int_0^h p_T dz = p_h h + \frac{1}{2} \rho g h^2$$

$u$  independent of  $z$   
and  $\rho = \text{constant}$ ,

$$\int_0^h \rho u^2 dz = \rho u^2 h$$

Using  $\int_0^h p_T dz = p_h h + \frac{1}{2} \rho g h^2$  and  $\int_0^h \rho u^2 dz = \rho u^2 h$

$$\int_0^h [\rho u^2 + p_T]_A dz = \int_0^{h-d} [\rho u^2 + p_T]_B dz + p_c d \quad \rightarrow$$

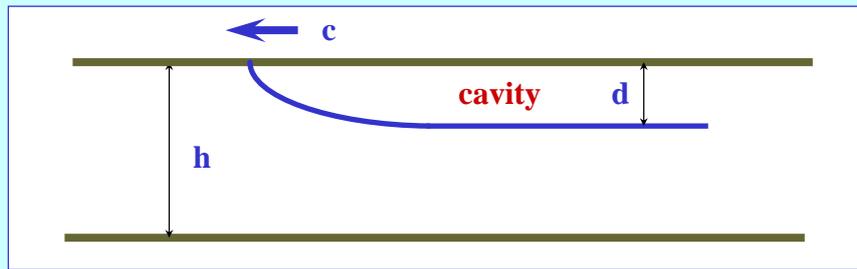
$$p c^2 h + \frac{1}{2} \rho g h^2 + \rho_A h = \rho U^2 (h-d) + \frac{1}{2} \rho g (h-d)^2 + p_c h$$

Continuity of mass (volume) implies that:  $ch = U(h-d)$

Recall that  $p_A + \frac{1}{2} \rho c^2 = p_c$

Then

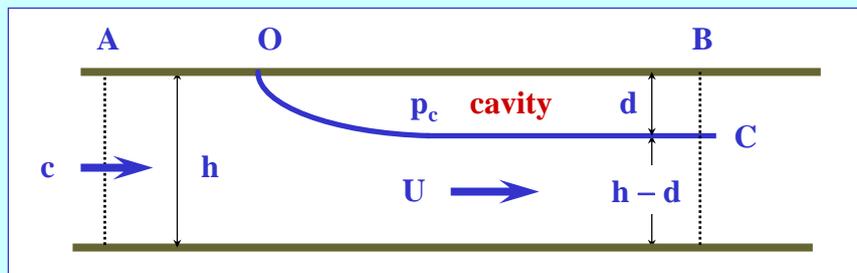
$$c^2 = g d \left[ \frac{2h-d}{h} \right] \left[ \frac{h-d}{h+d} \right] \quad \text{and} \quad U^2 = g d \left[ \frac{2h-d}{h^2-d^2} \right] h$$



- For a channel depth  $h$ , a cavity of depth  $d$  advances with speed  $c$  given by

$$c^2 = gd \left[ \frac{2h-d}{h} \right] \left[ \frac{h-d}{h+d} \right]$$

- Note that, as  $d/h \rightarrow 0$ ,  $c^2/(gd) \rightarrow 2$ , appropriate to the case of a **shallow cavity**.



- Suppose that the flow behind the cavity is **energy conserving**.
- Then we can apply Bernoulli's theorem along the free streamline from O to C, whereupon

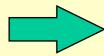
$$\cancel{p_c} + \cancel{\rho gh} = \cancel{p_c} + \frac{1}{2} \rho U^2 + \rho g (h-d)$$

➡  $U^2 = 2gd$

$$U^2 = 2gd \quad \text{and} \quad U^2 = gd \left[ \frac{2h-d}{h^2-d^2} \right] h$$

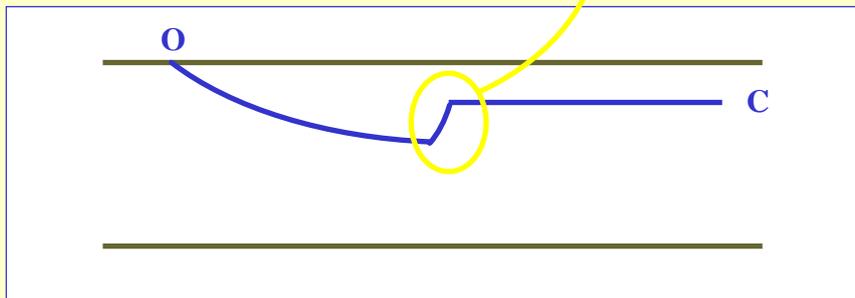
$$\rightarrow d = \frac{1}{2} H$$

Then  $c^2 = gd \left[ \frac{2h-d}{h} \right] \left[ \frac{h-d}{h+d} \right] \rightarrow c^2 = \frac{1}{2} gd$



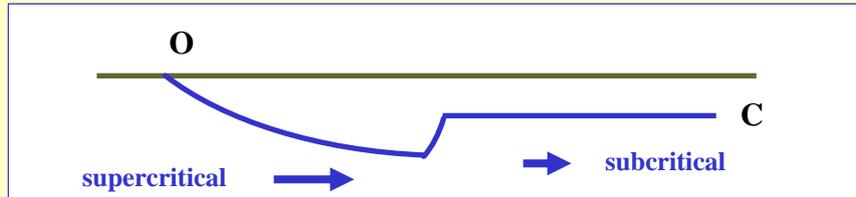
In an energy conserving flow, the cavity has a depth far downstream equal to one half the channel depth.

- If the flow is not energy conserving, there must be a jump in the stream depth behind the cavity.



Cavity flow with hydraulic jump

- According to the hydraulic jump theory, energy loss occurs at the jump and there must be a **loss of total head**, say, along the streamline **O** to **C**.



Then  $H_O = p_c + \rho gh$      $H_C = p_c + \frac{1}{2}\rho U^2 + \rho g(h - d)$

$$H_1 - H_2 = \rho\chi$$

➡  $p_c + \rho gh = p_c + \frac{1}{2}\rho U^2 + \rho g(h - d) + \rho\chi,$

or  $-U^2 + gd = \chi.$

$$U^2 = gd \left[ \frac{2h - d}{h^2 - d^2} \right] h \quad \rightarrow$$

$$\frac{d(h - 2d)}{h^2 - d^2} = \frac{2\chi}{gd} > 0$$

➡  $d < \frac{1}{2}h$     as expected

When the **cavity flow** is turned upside down, it begins to look like the **gravity-current configuration** - the jump and corresponding energy loss is analogous to the turbulent mixing region behind the gravity-current head.

- The foregoing theory can be applied to a gravity current of **heavy fluid** of density  $\rho_2$  moving into **lighter fluid** of density  $\rho_1$  if we **neglect the motion** within the heavier fluid.
- Then,  $g$  must be replaced by the **reduced gravity**

$$g' = g \frac{(\rho_2 - \rho_1)}{\rho_1}$$

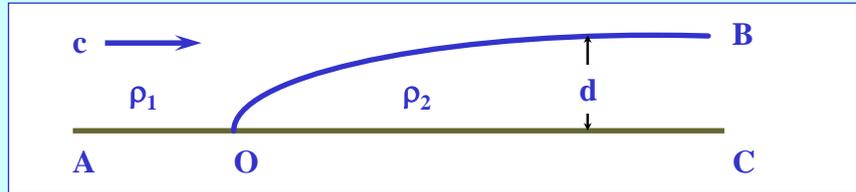
### The deep fluid case

- The case of a **shallow gravity current moving in a deep layer of lighter fluid** cannot be obtained simply by taking the limit as  $d/h \rightarrow 0$ .
- This would imply an **infinite energy loss** according to the foregoing theory.
- **Von-Kàrmàn** considered this case and obtained the same speed  $c$  that would have been obtained by taking the limit  $d/h \rightarrow 0$ ;


 $c / \sqrt{g'd} = \sqrt{2}$

- Although this result is correct, **von-Kàrmàn's derivation was incorrect** as pointed out by Benjamin (1968).
- I will consider **von-Kàrmàn's method** before Benjamin's.

## Deep fluid gravity current



➤ **Assumptions:**

- there is no flow in the dense fluid
- the pressure is hydrostatic and horizontally uniform

$$p_O = p_C = p_B + g\rho_2 d$$

- **Von-Kàrmàn applied Bernoulli's theorem between O and B (equivalent to the assumption of energy conservation) =>**

$$p_O = p_B + \frac{1}{2}\rho_1 c^2 + g\rho_1 d$$

**Eliminate the pressure difference  $p_O - p_B$  using**

$$p_O = p_C = p_B + g\rho_2 d$$

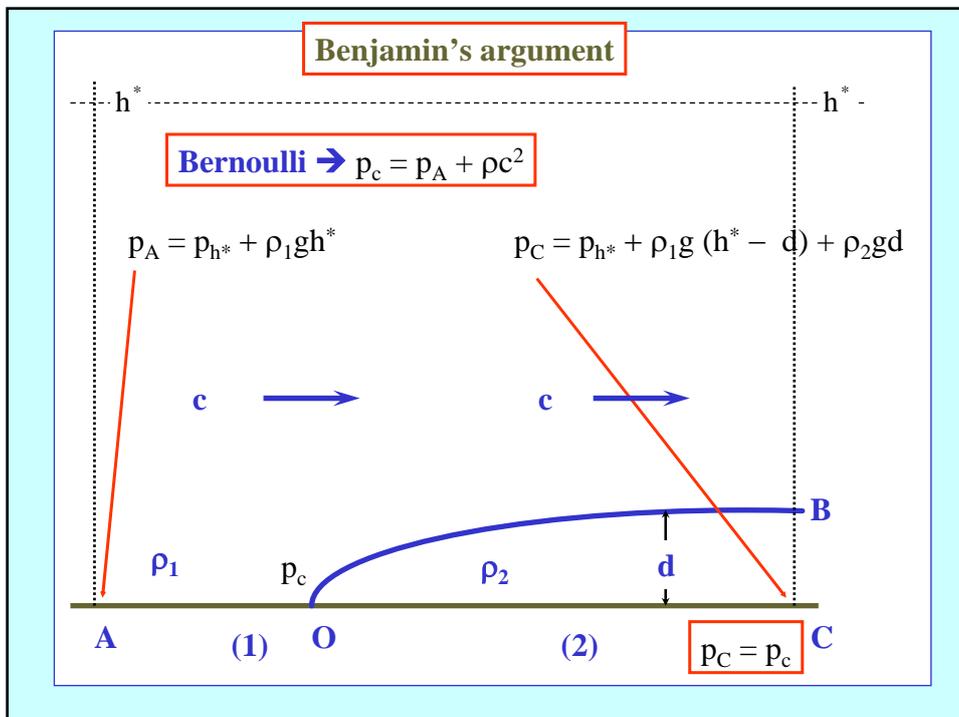
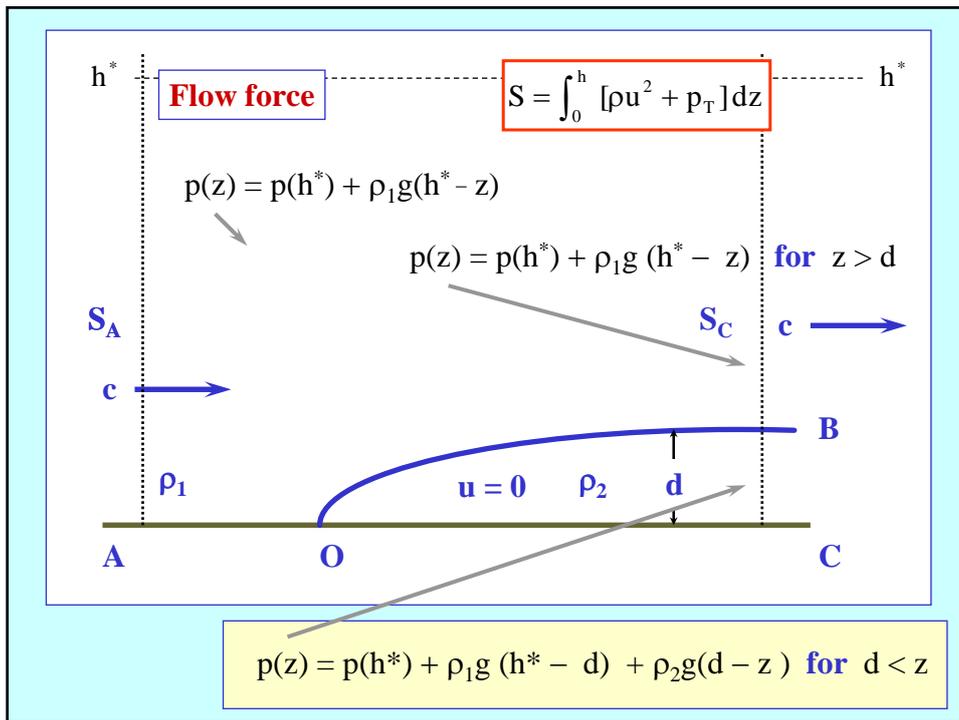
$$p_O = p_B + \frac{1}{2}\rho_1 c^2 + g\rho_1 d$$



$$c^2 = 2gd \frac{(\rho_2 - \rho_1)}{\rho_1}$$

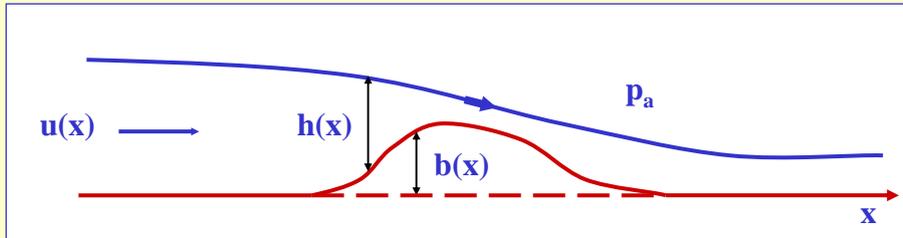
- **Benjamin (1968) pointed out that the assumption of energy conservation is inconsistent with that of steady flow in this problem, because there is a net force on any control volume enclosing the point O and extending vertically to infinity.**
- **The net force is associated with the horizontal pressure gradient that results from the higher density on the right of the control volume.**

The idea ...



## Flow over orography

- Consider the steady flow of a layer of non-rotating, homogeneous liquid over an obstacle



- Assume that the streamline slopes are small enough to neglect the vertical velocity component in comparison with the horizontal component =>

- Bernoulli's theorem gives for the free surface streamline

$$p_a + \frac{1}{2} \rho u^2 + \rho g (h + b) = \text{const } t$$

$$p_a + \frac{1}{2} \rho u^2 + \rho g [h + b(x)] = \text{const } t$$



$$e(x) = \frac{u^2}{2g} + h = -b(x) + \text{const } t$$

Defines the **specific energy**

$$\text{Continuity} \Rightarrow uh = Q = \text{constant}$$

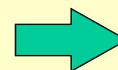
the **volume flux**  
per unit span

Can express **e** as a function of **h**



$$e = e(h) = \frac{Q^2}{2gh^2} + h$$

A graph of this function is shown in the next picture



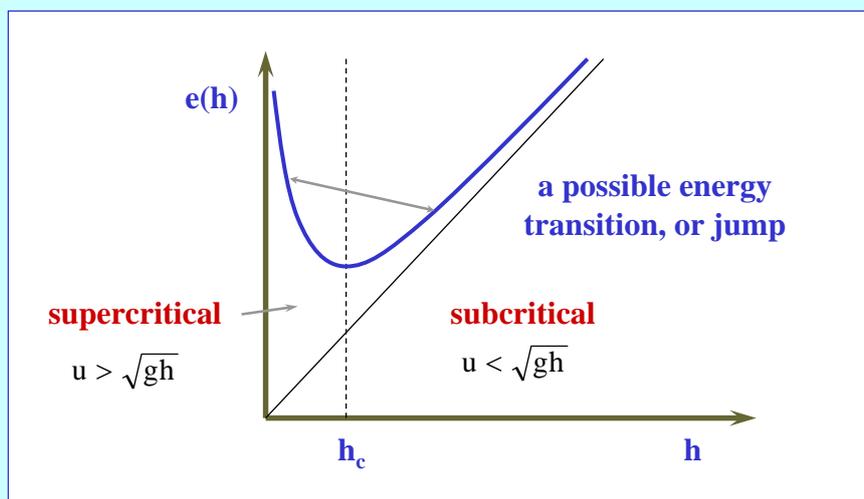
**Differentiating**  $e = \frac{Q^2}{2gh^2} + h$   $\rightarrow$   $\frac{de}{dh} = 1 - \frac{Q^2}{gh^3}$

$\frac{de}{dh} = 0$  **when**  $Q^2 = gh_c^3$   $\rightarrow$   $u^2 = gh_c$

**For a given energy  $e(h) > e(h_c)$ , there are two possible values for  $h$ , one  $> h_c$  and one  $< h_c$ .**

➤ **Given the flow speed  $U$  and fluid depth  $H$  far upstream where  $b(x) = 0$ ,  $Q = UH$  and**

$e = \frac{Q^2}{2gh^2} + h$   $\rightarrow$   $\frac{Q^2}{2g} \left[ \frac{1}{h^2} - \frac{1}{H^2} \right] + h - H = -b(x)$



**Regime diagram for flow over an obstacle**

$$\frac{Q^2}{2g} \left[ \frac{1}{h^2} - \frac{1}{H^2} \right] + h - H = -b(x)$$

- This may be solved for  $h(x)$  given  $b(x)$  as long as there are no jumps in the flow =>
- e.g. if  $h(x) > h_c$  for all values of  $x$ , in other words if the flow remains **subcritical**.
- If the flow is anywhere **supercritical**, there arises the possibility that hydraulic jumps will occur, leading to an abrupt transition to a subcritical state.
- The possibilities were considered in a series of laboratory experiments by Long (1953). See also Baines (1987).

**The End**